AMENDMENT

IN THE SPECIFICATION:

Please amend the second paragraph on page 4 as follows:

Referring to Figures 1 and 2 of the drawings, the illustrated disc brake has a fixed carrier 1 which carries a pair of friction ~~pads~~ elements 2 and [[,]]3, such as friction pads, disposed respectively at either side of a brake disc 4. The carrier 1 serves to mount the brake on a vehicle and to absorb torque sustained by the pads during a braking operation. A clamp member or caliper 5 straddles the brake disc and is mounted on the carrier 1 so as to be slidable axially of the brake disc relative to the carrier 1, by way of pins 6, in conventional manner. The caliper 5 carries an integral housing 7 which is adapted to mount a conventional air or other power actuator (not shown) on an external face 8 thereof. The housing 7 defines a chamber 9 within which a pivotal brake actuating lever 10 may conveniently perform an angular reciprocal swinging movement, as indicated by the arrows (Figure 2), under the action of a thrust member of the power actuator which, with the latter mounted on the external face 8, extends through an opening 11 of the housing 7 into engagement with a recess 12 of the brake actuating lever 10. The brake actuating lever 10 is integral with or attached to a rotary actuating member 13 which is rotatably supported within the caliper by way of a pair of needle bearing assemblies 14. The rotary actuating member 13 is recessed to house respective cylindrical rollers 15 and [[,]]16, the axes of which are offset from the rotary axis of the rotary actuating member 13 to form an eccentric actuating arrangement with the cylindrical rollers 15 and 16 bearing against respective thrust members 17A and 18A of adjacent adjustable tappet assemblies ~~indicated generally at~~ 17 and 18. Rotation of the brake actuating lever 10 and its connected rotary actuating ~~member~~shaft 13 causes actuating thrust to be applied via the tappet assemblies to the directly actuated friction element 2, and[[,]] by a reaction via the caliper 5, to the indirectly actuated friction element 3.

Please amend the first paragraph on page 5 as follows:

The thrust assemblies, illustrated as adjustable tappet assemblies 17 and 18, are disposed at either side of a ~~centercentre~~ line of the brake passing through the brake actuating lever 10 and are associated with an adjuster assembly 19 which lies laterally offset from and adjacent the adjustable tappet assembly 18. The adjuster assembly may be of any appropriate conventional type needing no detailed description for the purpose of the present invention. The adjuster responds to excessive movement of the friction elements 2 and [[,]]3 and produces resultant rotation of an adjuster shaft 20 which, via an output gear 21 and an intermediate idler gear 21A, rotates a pair of input gears 22 and [[,]]23 associated respectively with the adjustable tappet assemblies 17 and[[,]] 18.

Please amend the second paragraph on page 5 and bridging to page 6 as follows:

The adjustable tappet assemblies 17 and 18 are of identical construction and operation, and only the adjustable tappet assembly 17 will be described, with reference to Figures 1 and 3, in sufficient detail for a full understanding of the present invention. This assembly has an outer sleeve 24 which ~~is internally threaded at~~includes internal threads 24A and receives a hollow internal shaft 25 having an externally threaded portion 25A extending over a part of its length for cooperation with the internal thread of the outer sleeve 24. The hollow internal shaft 25 and the outer sleeve 24 form between them an adjuster strut of variable length. The hollow internal shaft 25 is provided, at its outer end, with a tappet head 26, which bears against the adjacent friction element 2 and which is releasably coupled to the shaft so as, conveniently, to be freely rotatable relative to the latter. To enable the hollow internal shaft 25 to be moved axially by rotation of the outer sleeve 24 so as to extend the adjuster strut in compensation for wear of the friction elements, it is necessary to lock this shaft against rotary movement. This is achieved, in the present embodiment, by providing the thrust member 17A, which is engaged over the adjacent end of the outer sleeve 24, with an elongate stem 27 of non circular cross-section which extends within the hollow internal shaft 25, of which at least a part 25B of the internal surface forms a complementary bore. The thrust members 17A and 18A of the respective tappet assemblies are each provided with, for example, a generally planar external surface for locking engagement with a corresponding adjacent surface on a fixed plate 28.

Please amend second paragraph on page 7 as follows:

It will be understood that the ~~seals annular lip~~ 34 and the further lip seal 37 may be of any convenient form and retained in any convenient manner on their respective ~~supports support~~ element 31 and metal retainer[[,]] 35 which, in turn, may have any convenient shape whilst retaining their essential purpose according to the invention.

Please amend the third paragraph on page 7 as follows:

The flat annular ~~portion~~ base 32 may conveniently provide stop means for limiting the axial movement of the tappet assembly at extremes of adjustment.